

Behavior of an Adjustable Bolted Steel Plate Connection during Field Installation

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ABSTRACT

This paper presents a bolted steel plate connection to join steel members at a range of angles with the capability of adjusting in-situ to accommodate additional angles or tolerances through cold bending. The connection features plates that are pre-bent (cold bent via a press brake) to defined angles, and then further cold bent during field installation (by bolt tightening) until turn-of-nut criteria are met. This approach uses a small number of unique components to facilitate prefabrication and rapid erection. Geometric studies were performed to select connection parameters for greatest adaptability to manufacturing/erection tolerances and versatility of member dimensions. A total of 13 scenarios were tested under field installation conditions to investigate the effect of the (1) bolt tightening procedure, (2) amount and direction of field bending, and (3) plate angle on surface strains. Strains were measured using Digital Image Correlation - an optical technique that captures full-field data. This paper presents a novel approach for bolted steel connections, measures the impact of field installation on surface strains, and makes implementation and design recommendations.

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18 **Author Keywords:** Bolted steel connection; Cold bending; Prefabrication; Rapid erection; Digital
19 Image Correlation

20 **INTRODUCTION**

21 This paper presents a novel approach for rapid erection of steel structures using prefabricated,
22 bolted connections that form moment-resisting joints between structural members in double shear
23 at a range of angles. The connection is adjustable, meaning that it is capable of changing angle in-
24 situ to accommodate additional angles or manufacturing and erection tolerances. More specifically,
25 connection plates are prefabricated by cold bending (via a press brake) to specific angles forming a
26 kit-of-parts comprised of a small number of unique components that can be used for a wide variety
27 of structural systems. For a given structure, these plates are then further cold bent during field
28 installation (via bolt tightening) until turn-of-nut criteria are met. Figure 1 shows a connection
29 between wide flange sections. Advantages of this approach include reduced cost and construction
30 time as prefabricated components can be used to form a wide variety of angled connections while
31 also allowing for erection tolerances. This approach can be implemented for any moment-resisting
32 joint between angled structural members in buildings (*e.g.*, apex connections of portal frames) and
33 bridges (*e.g.*, angled connections of arch and truss bridges). This is the first investigation of cold
34 bending for a kit-of-parts adjustable steel connection. The focus of this paper is on the geometric
35 development of the connection and measuring the surface strains induced during field installation.
36 This research is undertaken for a connection between flanges of wide flange structural members,
37 but other connection orientations/section shapes are possible. In addition to the development of
38 the adjustable connections, this research is relevant to double shear connections using cold bent
39 plates in general and is useful in assessing their behavior, as well as setting bend tolerances for
40 fabrication.

41 Cold bending is an appealing strategy to achieve adjustability as it offers cost and time sav-
42 ings, as opposed to heat-assisted bending (FHWA, 2015c), and can be readily performed in the
43 field. The implementation of cold bending for thin structural sections is well established in the
44 building industry, including implementations as early as the 1850s and design standards dating

45 from 1946 (Yu et al., 1996). However, applications are typically limited to structural members
46 (Yu and LaBoube, 1997; Davies, 2000; Hancock, 2003) with research involving connections pri-
47 marily focused on thin-walled fastener connections (Yu and LaBoube, 1997; Hancock, 2003), in
48 addition to bolted lap splices and a few other types (Chung and Lau, 1999; Pedreschi et al., 1997).
49 Cold bending (for bend radii exceeding $5t$, where t is the thickness) has only been permitted in
50 the bridge industry since a 2012 code revision, based on the findings of a Texas Department of
51 Transportation study (Keating and Christian, 2012; AASHTO, 2012). Cold bending has been used
52 in bridges including dapped girders (Keating and Christian, 2012; TXDOT, 2015), curved girder
53 bridges (Gergess and Sen, 2005a,b, 2008, 2009), a gussetless truss bridge (Cota and Zoli, 2012),
54 and connections for large skew bridges (HNTB Corporation, Genesis Structures, Inc., Structural
55 Engineering Associates, Iowa State University, 2014).

56 The primary benefit of this type of connection is adjustability, both in terms of connecting
57 members at different angles and accommodating manufacturing/erection tolerances. In conven-
58 tional arch or truss bridges, these connections could join angled members, thereby avoiding gusset
59 plates [*e.g.*, gussetless truss bridge (Cota and Zoli, 2012)]. Further, these connections could be
60 featured in modular bridges [*e.g.*, Pratt truss or network tied arch concepts proposed in Gerbo
61 et al. (2016a)] to reduce construction time. In a building environment, these connections could
62 join members of steel portal frames. The power of the connection lies in the kit-of-parts approach
63 which can be used for a wide variety of structural components.

64 **OBJECTIVES AND SCOPE**

65 The objective of this research is to develop a versatile adjustable bolted steel plate connection
66 and to investigate the behavior of this connection during field installation. A geometric investi-
67 gation of the adjustable plate connection was performed to select parameters for manufacturing
68 and erection tolerances as well as versatility of member dimensions. The authors have previously
69 investigated the behavior of the connection during the prefabrication process (Gerbo et al., 2016b).
70 In this prior work, full-field three-dimensional (3D) residual surface strains induced during cold
71 bending (via a press brake) were measured using Digital Image Correlation (DIC) and compared

72 with finite element predictions. The research presented in this paper focuses on the surface strains
73 induced in the connection during field installation (*i.e.*, cold bending via bolt tightening). A total
74 of 13 scenarios were tested under field installation conditions, with full-field 3D surface strains
75 measured using DIC, to investigate the effect of the (1) bolt tightening procedure, (2) amount and
76 direction of field bending, and (3) plate angle on surface strains. This paper develops a novel con-
77 cept for an adjustable bolted steel plate connection, measures surfaces strains induced during field
78 installation, and makes recommendations for design and implementation.

79 **GEOMETRIC INVESTIGATION OF CONNECTION PARAMETERS**

80 An extensive investigation to determine optimal geometric parameters (*e.g.*, plate length, initial
81 plate angles, bend radii, bolt hole type, member flange thickness, member depth, and connection
82 angle) was performed. Throughout this paper, the term “plate” refers to the plate connectors be-
83 tween the members. These plates connect flanges of members, which are referred to as simply
84 “member.”

85 **Geometric Parameters**

86 The adjustable connection is defined by the geometric parameters in Table 1 and shown in
87 Figure 2. For this study, the top and bottom plates are assumed to be identical (aside from their
88 width). However, the equations provided are expressed in general terms such that a designer could
89 choose different length, angle, and radii of curvature of the top and bottom plates (note that the
90 thickness and hole spacing are assumed to be the same in the top and bottom plate).

91 The pre-bent top and bottom plate angles ($\gamma = \beta$) are chosen to join a range of shallow angled
92 connections. The member angle (α) is considered for ranges of up to 5° greater than or less than
93 the pre-bent plate angles. A variety of top and bottom plate lengths ($l_1 = l_2$) and two different
94 radii of curvature for the top and bottom plates ($r_t = r_b$) were investigated. The plate thickness
95 (t_s) is selected to be on the order of half of the member flange thickness (t_m) for the considered
96 standard rolled wide flange members (W8, W10, and W12) with depths, d_m . This is an appropriate
97 proportion for the proposed double-shear connection. The hole sizes in the member (d_{mh}) are the
98 maximum allowable hole sizes for oversize, short slot, and long slot types for the selected bolt

99 diameter (d_b) per design code (AASHTO, 2014). Only oversized holes are considered for the holes
100 in the plate (d_{ph}) as it will be in direct contact with the bolt head and nut. Oversized, short or long
101 slots are necessary for the bolt up procedure. Note that the use of slotted holes in girder splice type
102 connections is not currently allowed by Code (AASHTO, 2014). This research does not address
103 the effect of hole size on the ultimate strength of the connection, and this will be the focus of future
104 work. The end distance between the bolt hole centerline and the edge of the plate (l_3) and the edge
105 of the member (l_4) is held constant. This is chosen to be more than the minimum edge distance
106 and less than the maximum edge distance prescribed by Code (AASHTO, 2014).

107 A study was performed to determine an optimized combination of the parameters. The pa-
108 rameters investigated include the plate lengths ($l_1 = l_2$), initial plate angles ($\gamma = \beta$), plate radii
109 of curvature ($r_t = r_b$), member slot type (d_{mh}), member thickness (t_m), member depth (d_m), and
110 connection angle (α) using the values provided in Table 1. The following sections first define
111 feasibility of a combination of parameters and then discuss the parametric investigation.

112 **Determining Feasibility of a Combination of Geometric Parameters**

113 A feasible combination of geometric parameters is defined as one for which a bolt can pass
114 through holes in the top plate, member, and bottom plate (*i.e.*, no interference between the bolt and
115 locations A-L in Figure 2D). To determine its feasibility, a comprehensive search of bolt locations
116 was performed for all angles and lateral positions of the bolt. Starting with the bolt in a vertical
117 orientation (*i.e.*, parallel to connection centerline) and centered on the member hole, the clearances
118 between the bolt and locations A-L were calculated. The angular orientations range from where
119 the bolt is parallel to the member in either direction (a range of 180° in increments of 1°). All
120 lateral locations of the bolt are considered from the furthest left to the furthest right of locations
121 A-L [in increments of 0.397 mm (0.0156 in.)].

122 Bolt clearances were determined by calculating the coordinates of locations A-L. Then, the
123 amount of clearance (c) between the location and the bolt is:

$$c = |\vec{v}| \sin(\omega) \quad (1)$$

124 where \vec{v} is the vector from the bolt edge line (\vec{u}) to the location, and ω is the angle between these
 125 vectors which can be found as follows:

$$\omega = \sin^{-1} \left(\frac{|\vec{u} \times \vec{v}|}{|\vec{u}||\vec{v}|} \right) \quad (2)$$

126 The clearance is calculated for locations A-L (equations provided in the following sub-section).
 127 Note that on the left side of the bolt \vec{u} is drawn pointing upward, and on the right side of the bolt
 128 \vec{u} is drawn pointing downward. A positive value of c indicates available clearance and a negative
 129 value represents lack of clearance (*i.e.*, interference) between the bolt and plates.

130 *Equations for Locations of Interference*

131 Equations for locations A-L are provided below, with subscripts x referring to the horizontal
 132 coordinate and y to the vertical coordinate with respect to the origin in Figure 2. All variables are
 133 defined in Table 1 and Figure 2. Angles in the equations are in units of radians and should be less
 134 than $\pi/2$. The equations shown here are for the top flange of the member, on the right side of the
 135 centerline. Analogous equations are used for the other locations.

136 The coordinates of locations A-D on the top plate are as follows:

$$\begin{aligned} A_x &= l_5 \cos \gamma + \frac{t_s}{2} \sin \gamma - \frac{d_{ph}}{2} \cos \gamma; & A_y &= -\left(l_5 - \frac{t_s}{2} \tan \gamma\right) \sin \gamma + t_s \cos \gamma + v_1 + \frac{d_{ph}}{2} \sin \gamma \\ B_x &= l_5 \cos \gamma + \frac{t_s}{2} \sin \gamma + \frac{d_{ph}}{2} \cos \gamma; & B_y &= -\left(l_5 - \frac{t_s}{2} \tan \gamma\right) \sin \gamma + t_s \cos \gamma + v_1 - \frac{d_{ph}}{2} \sin \gamma \\ C_x &= l_5 \cos \gamma - \frac{t_s}{2} \sin \gamma - \frac{d_{ph}}{2} \cos \gamma; & C_y &= -\left(l_5 - \frac{t_s}{2} \tan \gamma\right) \sin \gamma + v_1 + \frac{d_{ph}}{2} \sin \gamma \\ D_x &= l_5 \cos \gamma - \frac{t_s}{2} \sin \gamma + \frac{d_{ph}}{2} \cos \gamma; & D_y &= -\left(l_5 - \frac{t_s}{2} \tan \gamma\right) \sin \gamma + v_1 - \frac{d_{ph}}{2} \sin \gamma \end{aligned} \quad (3)$$

137 where l_5 is the distance from the centerline to the top plate hole along the plate axis:

$$l_5 = \left(\frac{l_1}{2} - l_3 \right) + \left(r_t + \frac{t_s}{2} \right) (\tan \gamma - \gamma) \quad (4)$$

138 Length v_1 is measured from the origin to the extension of the plate as drawn. This is different for

139 each contact type (Figure 2E) for the top plate (T):

Type T1 if: $\alpha \geq \gamma$ and $r_t \sin \gamma \geq g$

Type T2 if: $\alpha \geq \gamma$ and $r_t \sin \gamma < g$ (5)

Type T3 if: $\alpha < \gamma$

140 Length v_1 can be found as:

$$v_1 = \begin{cases} (r_t \sin \gamma) \tan \gamma - r_t \cos \left(\sin^{-1} \left(\frac{g}{r_t} \right) \right) - \cos \gamma & \text{if T1} \\ g \tan \gamma & \text{if T2} \\ r_t \sin \gamma \tan \gamma + v_2 - (r_t \sin \gamma + h_1 - g) \tan \alpha & \text{if T3} \end{cases} \quad (6)$$

141 where the vertical (v_2) and horizontal (h_1) dimensions of the straight portion of the top plate are:

$$v_2 = \left[\frac{l_1}{2} + \left(r_t + \frac{t_s}{2} \right) (\tan \gamma - \gamma) - \tan \gamma \left(r_t + \frac{t_s}{2} \right) \right] \sin \gamma \quad (7)$$

$$h_1 = \left[\frac{l_1}{2} + \left(r_t + \frac{t_s}{2} \right) (\tan \gamma - \gamma) - \tan \gamma \left(r_t + \frac{t_s}{2} \right) \right] \cos \gamma \quad (8)$$

142 The coordinates of locations E-H on the member are as follows:

$$\begin{aligned} E_x &= -g - l_4 \cos \alpha + t_m \sin \alpha - \frac{d_{mh}}{2} \cos \alpha; & E_y &= -l_4 \sin \alpha + t_m \sin \alpha + \frac{d_{mh}}{2} \sin \alpha \\ F_x &= -g - l_4 \cos \alpha + t_m \sin \alpha + \frac{d_{mh}}{2} \cos \alpha; & F_y &= -l_4 \sin \alpha + t_m \sin \alpha - \frac{d_{mh}}{2} \sin \alpha \\ G_x &= -g - l_4 \cos \alpha - \frac{d_{mh}}{2} \cos \alpha; & G_y &= -l_4 \sin \alpha + \frac{d_{mh}}{2} \sin \alpha \\ H_x &= -g - l_4 \cos \alpha + \frac{d_{mh}}{2} \cos \alpha; & H_y &= -l_4 \sin \alpha - \frac{d_{mh}}{2} \sin \alpha \end{aligned} \quad (9)$$

143

The coordinates of locations I-L on the bottom plate are as follows:

$$\begin{aligned}
 I_x &= l_6 \cos \beta + \frac{t_s}{2} \sin \beta - \frac{d_{ph}}{2} \cos \beta; & I_y &= -\left(l_6 - \frac{t_s}{2} \tan \beta\right) \sin \beta + t_s \cos \beta - v_3 + \frac{d_{ph}}{2} \sin \beta \\
 J_x &= l_6 \cos \beta + \frac{t_s}{2} \sin \beta + \frac{d_{ph}}{2} \cos \beta; & J_y &= -\left(l_6 - \frac{t_s}{2} \tan \beta\right) \sin \beta + t_s \cos \beta - v_3 - \frac{d_{ph}}{2} \sin \beta \\
 K_x &= l_6 \cos \beta - \frac{t_s}{2} \sin \beta - \frac{d_{ph}}{2} \cos \beta; & K_y &= -\left(l_6 - \frac{t_s}{2} \tan \beta\right) \sin \beta - v_3 + \frac{d_{ph}}{2} \sin \beta \\
 L_x &= l_6 \cos \beta - \frac{t_s}{2} \sin \beta + \frac{d_{ph}}{2} \cos \beta; & L_y &= -\left(l_6 - \frac{t_s}{2} \tan \beta\right) \sin \beta - v_3 - \frac{d_{ph}}{2} \sin \beta
 \end{aligned} \tag{10}$$

144

where l_6 is the length from the centerline to bottom plate hole centerline along the axis of the plate:

$$l_6 = \left(\frac{l_2}{2} - l_3\right) + \left(r_b + \frac{t_s}{2}\right)(\tan \beta - \beta) \tag{11}$$

145

Length v_3 is measured from the origin to the extension of the plate as drawn. This is different for

146

each contact type (Figure 2E) for the bottom plate (B):

Type B1 if: $\alpha \leq \beta$, $g - t_m \sin \alpha \leq (r_b + t_s) \sin \beta$, and $\lambda \geq \alpha$

Type B2 if: $\alpha \leq \beta$ and $g - t_m \sin \alpha > (r_b + t_s) \sin \beta$

Type B3 if: $\alpha > \beta$

Type B4 if: $\alpha \leq \beta$, $g - t_m \sin \alpha \leq (r_b + t_s) \sin \beta$, and $\lambda < \alpha$

(12)

147

where λ is the angle from the center of curvature of the bottom plate to the point of contact with

148

the member:

$$\lambda = \sin^{-1} \left(\frac{g - t_m \sin \alpha}{r_b + t_s} \right) \tag{13}$$

149

Length v_3 can be found as:

$$v_3 = \left\{ \begin{array}{ll} -r_b \cos \beta - r_b \sin \beta \tan \beta + \frac{t_m}{\cos \alpha} + v_4 & \text{if B1} \\ t_m \cos \alpha + \frac{t_s}{\cos \beta} - (g - t_m \sin \alpha) \tan \beta & \text{if B2} \\ \frac{t_s}{\cos \beta} + v_5 + \frac{t_m}{\cos \alpha} - v_6 & \text{if B3} \\ -r_b \cos \beta - r_b \sin \beta \tan \beta + t_m \cos \alpha + v_7 + (r_b + t_s) \cos \alpha & \text{if B4} \end{array} \right\} \quad (14)$$

150 where v_4 is the vertical distance from the center of curvature of the bottom plate to the member
 151 contact location for case B1. Length v_4 is defined as follows:

$$v_4 = \sqrt{(r_b + t_s)^2 - \left(\frac{g - t_m \sin \alpha}{2}\right)^2} \quad (15)$$

152 The vertical distance between the contact point and the bottom corner of the member (v_5) for
 153 contact case B3 is:

$$v_5 = \left(\frac{\frac{l_2}{2} + (r_b + \frac{t_s}{2})(\tan \beta - \beta)}{2} \cos \beta + \frac{t_s}{2} \sin \beta - g + t_m \sin \alpha \right) \tan \beta \quad (16)$$

154 The vertical dimension of the bottom plate (v_6) for contact case B3 is:

$$v_6 = \frac{\frac{l_2}{2} + (r_b + \frac{t_s}{2})(\tan \beta - \beta)}{2} \sin \beta \quad (17)$$

155 The vertical distance between the contact point and the bottom corner of the member (v_7) for
 156 contact case B4 is:

$$v_7 = \left((r_b + t_s) \sin \beta - g + t_m \sin \alpha \right) \tan \alpha \quad (18)$$

157 **Description of Parametric Investigation**

158 A parametric investigation was performed as follows:

159 *Level 1 - Connection Angle and Gap Analysis for Manufacturing and Erection Tolerances*

160 The first level varies the member angle (α) and gap (g) between the members in order to
 161 determine the range of member connection angles and the minimum and maximum gap that are
 162 feasible for a given configuration. It is advantageous for the connection to achieve the widest

163 range of member connection angles and to span the widest range of gaps between members to
164 accommodate erection tolerances on both the angular and lateral placement of members.

165 For every combination of parameters of α and g , the feasibility of the configuration was evalu-
166 ated for (1) member angles (α) plus or minus 5° of the pre-bent splice plate angles ($\gamma = \beta$) in 0.5°
167 increments and (2) gaps (g) between a lower-bound based on a selected minimum clearance (e)
168 and an upper-bound based on plate lengths ($l_1 = l_2$).

169 From a representative Level 1 analysis (Figure 3A), it is shown that with higher member angles
170 (α) the range of allowable gap (g) is reduced. As a measure of the erection versatility, the area
171 between the two lines indicating the minimum and maximum gap is calculated and recorded as
172 C_{vers} (shaded region in Figure 3A) to be used in upper level geometric analyses.

173 *Level 2 - Member Thickness and Depth for Versatility of Member Dimensions*

174 A second level analysis considers the sensitivity of C_{vers} to varying member thicknesses (t_m)
175 and member depth (d_m). This relates to the versatility of a design, allowing for the widest range of
176 member sizes for a given configuration.

177 A representative Level 2 analysis (Figure 3B) shows that with lower member flange thicknesses
178 (t_m) there is greater versatility (C_{vers}) than with higher member flange thicknesses. The considered
179 member depths (d_m) have little effect on versatility. The volume beneath the surface of this plot is
180 calculated and recorded as D_{vers} as a measure of the design versatility of the specified configuration
181 to be used in upper level analyses.

182 *Level 3 - Plate Lengths and Initial Angles*

183 The third level of analysis considers the metric D_{vers} for a variety of plate lengths ($l_1 = l_2$) and
184 initial plate angles ($\gamma = \beta$), as connections with higher angles require longer plates.

185 A representative Level 3 analysis (Figure 3C) shows that, up to a point, an increase in versatility
186 can be achieved by increasing plate length. This is because the longer plate lengths allow deeper
187 members to be connected without causing interference of the bottom flanges. To a lesser degree,
188 it is shown that increasing plate angles ($\gamma = \beta$) decreases versatility, as was expected because the
189 increased angles minimize available space for bolts to pass through the plated connection.

190 *Level 4 - Radii of curvature and Member Hole Types*

191 The fourth level analysis considers the radii of curvature ($r_t = r_b$) and member hole types
192 (d_{mh}). The radii of curvature considered were 63.5 mm (2.5 in.) and 102 mm (4 in.). The former
193 corresponds to the $5t$ minimum bend radii allowed by bridge design Code (AASHTO, 2012). The
194 member hole types considered include oversized holes, short slots, and long slots.

195 This analysis repeats the Level 3 studies 6 times to evaluate all considered combinations of
196 radii and member hole types. Results for $r_t = r_b = 102$ mm (4 in.) are shown in Figure 3D. Results
197 for the smaller bend radius are not included as it was found that the value for the radius of curvature
198 does not play a significant role in versatility. This is because in most configurations, the member
199 contacts the straight portion of the plate resulting in the radius of curvature having minimal impact
200 on the geometric analysis. It is shown that a significant improvement in versatility can be achieved
201 through the use of long slots in the member, but short slots are very similar to oversized holes.

202 **Results**

203 The geometric parameters of the connection investigated in the experimental program were
204 chosen based on the results of these studies. From the results of the Level 1 study (Figure 3A), it
205 was found that higher member angles result in more stringent gap ranges to achieve feasibility. In
206 the Level 2 study (Figure 3B), it was found that thicker member flanges result in reduced versatility,
207 but member depth had little impact on versatility. The Level 3 study (Figure 3C) indicates that
208 longer plates allow for deeper members to be connected by preventing interference of the bottom
209 flange. From the Level 4 study (Figure 3D), it was found that the considered radii of curvature
210 had little impact on the geometric analysis, and that longer slots in the member can dramatically
211 increase the connection's versatility.

212 Based on these studies the member hole type is taken as a long slot [$d_{mh} = 47.6$ mm (1.875
213 in.)] to ensure the widest variety of feasible geometry. A 102 mm (4 in.) radius of curvature (r_t
214 $= r_b$) was chosen as the radius does not significantly affect the versatility of the connection and
215 larger bend radii reduce the magnitude of residual strains from prefabrication. The plate lengths
216 ($l_1 = l_2$) were chosen for specific plate angles ($\gamma = \beta$) to achieve high versatility (D_{vers}). For $\gamma = \beta$

217 = 0° , $l_1 = l_2 = 381$ mm (15 in.); for $\gamma = \beta = 5^\circ$, $l_1 = l_2 = 432$ mm (17 in.); for $\gamma = \beta = 10^\circ$, $l_1 = l_2 =$
218 483 mm (19 in.); for $\gamma = \beta = 15^\circ$ plates, $l_1 = l_2 = 533$ mm (21 in.). Note that the standard threaded
219 length of A325 bolts can induce limitations for connections with large differences between the
220 plate angles ($\gamma = \beta$) and member angle (α), as they require a significant threaded length to fully
221 tighten the connection. Figure 4 shows the idealized geometry from this study and the as-built
222 implementation, verifying this geometric study and highlighting its robustness.

223 **EXPERIMENTAL PROGRAM**

224 A total of 13 connection scenarios were tested to investigate the effect of the (1) bolt tightening
225 procedure, (2) amount and direction of field bending, and (3) plate angle on the surface strains of
226 the plates induced during field installation (Table 2, Figure 4). Scenario 1 was tested three times to
227 demonstrate repeatability. All other scenarios were tested once.

228 Each scenario used three ASTM A36 steel plates to connect the top flanges of two W10x88
229 beams (Figure 5). A single top plate [12.7 mm (0.500 in.) thick by 203 mm (8.00 in.) wide
230 with lengths varying from 381 to 533 mm (15.0 to 21.0 in.)] connected the top surface of the
231 top flanges, while two bottom plates [12.7 mm (0.500 in.) thick by 76.2 mm (3.00 in.) wide
232 with lengths varying from 381 to 533 mm (15.0 to 21.0 in.)] connected the underside of the top
233 flanges, with one bottom plate located on each side of the web of the beams. The plates were
234 pre-bent via a press brake and the residual strains induced during prefabrication were reported
235 in Gerbo et al. (2016b). Each connection used four ASTM A325 19.1 mm (0.750 in.) diameter
236 bolts. The stress strain relationships of A36 plate and A325 bolt specimens, found according
237 to ASTM standards A370 and F606 respectively (ASTM, 2015, 2014), are shown in Figure 6.
238 Bolt testing was performed by Laboratory Testing Inc. (Laboratory Testing Inc., 2017). A325
239 bolts were selected as these are the most commonly used high-strength bolt (Salmon et al., 2009)
240 and feature enhanced ductility and lower susceptibility to stress corrosion and hydrogen stress
241 cracking when galvanized in comparison to higher strength bolts (*e.g.*, A490) (Kulak et al., 2001).
242 Further, the Federal Highway Administration (FHWA) recommends the use of A325 bolts over
243 A490 (FHWA, 2015a) and bridge design Code prohibits the use of galvanized A490 bolts (FHWA,

244 2015b; AASHTO, 2014).

245 Each W10x88 beam was supported by a W10x88 stub column connected to a W12x106 grade
246 beam that was anchored to the laboratory floor (Figure 5). Different stub columns were used to vary
247 the angle of the beams. Bolts were tightened manually via a torque wrench, with the assistance of a
248 torque multiplier (Figure 5B). One W6x12 cantilevered column was located at each end of the test
249 setup and bolted to the laboratory floor. These columns provided a reaction point for the tools used
250 to tighten the bolts and support the instrumentation system (Figure 5B and 5C). Figure 4 shows the
251 idealized geometry as well as the experimental setup for each tested scenario. In all scenarios the
252 bolts fit into the assembly as anticipated, verifying the accuracy of the geometric study.

253 The full-field surface strains in the plates were measured using DIC, a non-contact optical
254 technique. The 3D DIC system (provided by Trilion Quality Systems) consisted of two cameras
255 [2448 x 2050 pixels with 12.0 mm (0.472 in.) manual focus lenses] and utilized ARAMIS DIC
256 software to measure surface strains within the field-of-view (FOV). Multiple camera positions
257 and mirrors were used to capture the behavior of the top surface of the top plate and bottom
258 surfaces of both bottom plates (Figure 5C). The FOV for each position was approximately 610 by
259 510 mm (24.0 by 20.1 in.). Stereo pairs of photographic images were captured and divided into
260 regions called facets that are 13 by 13 pixels. Using photogrammetric triangulation and pattern
261 recognition, these facets were tracked through a series of images to produce 3D full-field surface
262 strains. Overall, the system is capable of measuring strains up to 0.0001 mm/mm (0.0001 in./in.)
263 (TRILION, 2016).

264 There were several challenges in the application of DIC to this research. In general, DIC
265 requires a clear view of any measured surface from both cameras. However, in this research the
266 bolt assemblies block a portion of the steel plates from view of the cameras, causing some minor
267 data loss. Camera positions were optimized to reduce data loss as much as possible. In addition,
268 the DIC pattern is typically achieved by painting the specimen with a white background and black
269 dots. In this research, the plates were bent using a press brake, resulting in surface abrasion that
270 traditional DIC paint could not withstand. Instead, the surface was first coated with CerMark

271 LMM-6000 Metal Laser Marking Spray (Ferro, 2016) and then etched with a durable random
272 pattern using a laser cutter (Universal Laser Cutter, VLS 6.60, 50W laser) as was done in Gerbo
273 et al. (2016b). This surface preparation resulted in significant specular reflection. This resulted in
274 challenges with the lighting during testing, as both cameras must not only be able to clearly see the
275 plate, but also must have similar lighting/reflection to enable the algorithm to correlate locations
276 between the two images. The authors recommend using careful lighting when specularly reflective
277 surfaces are tested.

278 **BEHAVIOR DURING FIELD BENDING**

279 **Effect of Bolt Tightening Procedure**

280 This section focuses on the impact of the bolt tightening procedure on the surface strains in-
281 duced in the plates during field installation. As shown in Table 2, five different bolt-tightening
282 procedures were investigated (Scenarios 1-5) when bending plates from $\gamma = \beta = 10^\circ$ to $\alpha = 17.5^\circ$.
283 The difference between plate angles ($\gamma = \beta$) and member angles (α) is defined as δ , and consid-
284 ered positive when the member angle is greater than the plate angles (*e.g.*, for Scenarios 1-5, δ
285 = $+7.5^\circ$). For each scenario, the connection was considered fully tightened when the turn-of-nut
286 criteria (AASHTO, 2010) was satisfied for each individual bolt.

287 Figure 7 shows a plan view of the top and bottom plates and indicates four longitudinal lines
288 for which data will be presented: line A is the centerline of bottom plate 1 (BP1), line B intersects
289 the top row of bolts for the top plate (TP), line C intersects the bottom row of bolts for the top
290 plate, and line D is the centerline of bottom plate 2 (BP2).

291 Figure 8 displays the measured circumferential surface strains (ϵ_x) as a function of the location
292 along lines A-D for Scenarios 1, 2, 4, and 5. The magnitudes of strains are very similar in both the
293 top and bottom plates [around 0.03 mm/mm (0.03 in./in.)] for all scenarios. As expected, the peak
294 strains in the top plate occur near the point of contact with the beams (shown as dashed vertical
295 lines) and have relatively narrow widths [approximately 30 to 40 mm (1.2 to 1.6 in)]. The peak
296 strains in the bottom plates occur at the edge of the pre-bent region (indicated by the gray-shaded
297 region). These peaks are on the edges of the pre-bent region which has been work hardened and

298 therefore has a higher yield strength as opposed to the straight portions of the plate. There are also
299 strains measured throughout the pre-bent region, as the bottom plates behave similar to a beam
300 under four-point bending.

301 The measured strains from the three tests of Scenario 1 (identified as Scenarios 1a, 1b, and 1c)
302 were very similar, demonstrating that the connection assembly and bolt tightening procedure are
303 repeatable. With repeatability demonstrated, only one test of all other scenarios was performed.

304 The left column of Figure 9 shows the progression of strain during the bolt tightening process of
305 Scenario 1, while the right column shows the final strain induced by the bolt tightening procedures
306 of Scenarios 1, 4, and 5 (criss-cross, clockwise, and counter-clockwise, respectively). Figure 9
307 (left) shows the full-field surface strains within the DIC FOV in the top and bottom plates for
308 Scenario 1, with the measured results shown after six turns of each bolt, at the point of contact
309 between the plates and the beams, and after the final turn of the bolts (when the turn-of-nut criteria
310 was satisfied). As expected, the strain in the top plate increases in magnitude as the bolts are
311 tightened. However, the net section of the bottom plate (near the bolt holes) experiences a peak
312 strain after six turns, then decreases in magnitude as the bolts are further tightened. During bolt
313 tightening, the bottom plate starts to bend and moves towards the member with minimal initial
314 deformations at the center of the bottom plate. Once the bottom plate comes into contact with the
315 member, the deformations at the center of the plate become more dominant and reach peak strain
316 after the final bolt turn. The hysteresis in the net section of the bottom plates during installation
317 must be accounted for during design, as it enhances the potential for reduced ductility and fatigue
318 resistance of the steel in the cold-worked region.

319 *Increment of Tightening*

320 Scenarios 1, 2, and 3 all used the criss-cross tightening pattern, but with varying increments (or
321 number) of turns at a time. While Scenario 1 and 2 resulted in very similar strain patterns, it was
322 observed in Scenario 2 that tightening in larger increments (three turns per tightening step) resulted
323 in noticeable gouging of the bolts. Scenario 3 (in which bolts were fully tightened individually)
324 is not plotted on Figure 8 because bolt 3 fractured during the tightening process. Therefore, it is

325 recommended only one full turn of an individual bolt at a time be implemented.

326 *Pattern of Tightening*

327 Scenarios 4 and 5 use clockwise and counter-clockwise tightening patterns, respectively, as
328 compared to Scenario 1, which uses the criss-cross pattern. The measured strains in Scenarios 1,
329 4, and 5 are very similar, as shown in Figure 8 and in Figure 9. However, there is some asymmetry
330 in the peak strains on the bottom plate which changes location based on the tightening pattern.
331 This asymmetry is more pronounced in the bottom plate as the bottom plates are restrained by
332 just two bolts and are therefore more susceptible to differences in the order in which bolts are
333 tightened, compared to the top plate. This is more pronounced in Scenario 4 and 5 which feature
334 circular tightening patterns as opposed to Scenario 1 which uses the criss-cross pattern. Further,
335 some strain bands occur near the net section of the top plate for Scenarios 4 and 5. This is not
336 desirable as it may reduce fatigue resistance of the connection. As a result, the criss-cross pattern
337 is recommended over circular patterns.

338 In general the peak strains are not significantly affected by tightening procedure, therefore the
339 recommended tightening procedure is one turn per increment, with a criss-cross tightening pattern.

340 **Effect of Amount and Direction of Field Bending**

341 This section investigates the impact of the amount (*i.e.*, number of degrees δ) and direction
342 (*i.e.*, increasing or decreasing the angle of the pre-bent plate) on the surface strains induced in the
343 connection. As shown in Table 2, four different member angles were investigated (Scenarios 1
344 and 6-8), with Scenario 1 serving as the baseline for comparison of the measured behaviors. Each
345 scenario used the same bolt tightening procedure as Scenario 1 (*i.e.*, one full turn of an individual
346 bolt at a time, using a repeated crisscross pattern to tighten the entire four bolt connection).

347 Figure 10 (left) shows the measured circumferential surface strains during field bending as a
348 function of the location along lines A-D and Figure 11 shows the full-field strains. As expected,
349 compressive strains developed in the top plate while tensile strains developed in the bottom plate
350 for Scenarios 1 and 6 (where $\delta > 0$). Similarly, for Scenarios 7 and 8 (where $\delta < 0$), compressive
351 strains developed in the bottom plate while tensile strains developed in the top plate. Also as

352 expected, the highest absolute peak strains occurred when the magnitude of the field bend (δ)
353 was largest. The overall peak strain [approximately 0.035 mm/mm (0.035 in./in.)] is observed
354 in Scenario 1 in the top plate near the point of contact with the beams (shown as dashed vertical
355 lines). In Scenario 8, the peak strain [approximately 0.02 mm/mm (0.02 in./in.)] occurred in the
356 net section of the top plate. This is an important feature as the cold working here would reduce
357 the ductility of the plate at the net section (bolt holes) enhancing the potential for reduced fatigue
358 resistance of the steel in the cold-worked region. This was not observed in the plots in Figure 10
359 due to data loss along lines B and C (where the section cut goes through the holes in the plate as
360 shown in Figure 7). The bolt assembly also blocks a portion of the DIC view of the plate due to the
361 washers being larger diameter [37.0 mm (1.46 in.)] than the holes in the plates [23.8 mm (0.938
362 in.)].

363 The peak strains in the bottom plates occur at the edge of the pre-bent region for Scenario 1,
364 but occur in the center pre-bent region and near the line of contact with the beams for Scenario 8.
365 Scenario 8 creates a region of constant moment in between the point of contact with the member,
366 and thus the plateau in the center is expected. In Scenario 8, there are no double peaks in strain
367 around the pre-bent region, as observed in Scenario 1. This is due to Scenario 8 inducing bending
368 in the direction opposite the direction of prefabrication. Here, the Bauschinger effect is lowering
369 the yield stress in the pre-bent region. While the magnitude of this peak strain was smaller, the
370 distribution of plastic strains was much wider (covering the entire pre-bent region of the bottom
371 plate). Figure 11 shows that some minor strain banding occurs across the top plates outside the pre-
372 bent region, this is believed to be due to inhomogeneity in the grain structure of the steel. Scenarios
373 6 and 7 (featuring $\delta = +2.5^\circ$ and -2.5° field bends, respectively) had very small measured peak
374 strains [around 0.005 mm/mm (0.005 in./in.)], indicating that small field bends do not generate
375 significant surface strains. To minimize the induced strains, connections with $\delta = \pm 2.5^\circ$ or lower
376 are recommended.

377 The right column of Figure 10 shows the cumulative strains [*i.e.*, strains from field installation
378 plus residual strains from pre-fabrication]. These cumulative strains reach peak magnitudes of

379 approximately 0.07 mm/mm (0.07 in./in.) in the bottom plates of Scenario 1. The bottom plates of
380 Scenario 1 experience the highest cumulative strain because the induced strains from field bending
381 and prefabrication occur in the same region, and the strains are additive because $\delta > 0$. In the top
382 plates of Scenario 1, the strain induced during field bending is in a different location than the strain
383 from prefabrication, hence the three distinct peaks along lines B and C. Conversely, Scenario 8
384 experiences a decrease in magnitude of cumulative strain because $\delta < 0$.

385 Understanding the cumulative final strain and hysteresis induced are important design factors.
386 Previous studies have found that plastic strains up to 0.10 mm/mm (0.10 in./in.) resulted in minimal
387 effect on ductility and fracture toughness (Keating and Christian, 2012). The measured strains in
388 this study are below this upper limit. Fatigue behavior of steel is not only dependent on applied
389 cyclic load, but also on loading history (Erber et al., 1992). The plastic strains induced during
390 prefabrication and field installation will have an effect on components subjected to fatigue loading;
391 therefore, strain history must be taken into consideration during the design process. In many of
392 the tested scenarios, the locations of induced plastic strain are not coincident with critical areas of
393 the splice plates (*i.e.*, the net section across bolt holes), and are not likely to significantly affect
394 the overall design of the connection. The cumulative induced plastic strains will result in reduced
395 fracture toughness and ductility, including also the effect of strain aging (FHWA, 2015b). Strain
396 hysteresis produces additional micro-defects that can reduce fatigue life and must be considered
397 in design (FHWA 2015b, Erber et al. 1992). The quantitative impact on fatigue performance is an
398 area for future research.

399 **Effect of Varying Plate Angles**

400 This section investigates the effect of different plate angles. All scenarios discussed here use
401 the same bolt tightening procedure as Scenario 1. As shown in Table 2, four different plate angles
402 were investigated: Scenario 1, 9-13. All feature small bend angles ($\delta = \pm 2.5^\circ$) and therefore have
403 low strains [with peaks on the order of 0.01 mm/mm (0.01 in./in.)].

404 As shown in Figure 12, the peak magnitude of strain in Scenario 9 occurs in the top plate near
405 the point of contact with the beam on the right side. This asymmetry can be attributed to slight

406 differences in the height of members in the reaction frame. This scenario is particularly susceptible
407 to slight imperfections in the reaction frame as the plates are initially flat.

408 In Scenario 10, peak strains occur near the center (within the pre-bent region) of the bottom
409 plates. This is consistent with the behavior observed in Scenario 8 which also has a $\delta < 0$. Scenario
410 11, which has a $\delta > 0$, exhibits small strain concentrations in the top plate near the line of contact
411 with the member, as expected and consistent with Scenario 1.

412 Figure 13 displays the measured surface strains for Scenarios 12 and 13. In Scenario 12,
413 strains are mostly negligible. Scenario 13 experiences higher magnitudes of strain approaching
414 0.01 mm/mm (0.01 in./in.), with peaks in the bottom plates at the center, and in the top plates at
415 the lines of contact with the member. This pattern is consistent with that observed in Scenario 1.

416 Overall, this section demonstrates that varying the level of initial pre-bend has little impact on
417 the strains induced during field installation. Rather the amount and direction of bend, as investi-
418 gated in the prior section, are more important.

419 **RECOMMENDATIONS AND CONCLUSIONS**

420 This paper investigated an adjustable bolted steel plate connection to join a range of angled steel
421 members. The connection features pre-bent plates that are further bent during field installation via
422 bolt tightening. This research focused on the field installation process following prior work by
423 the authors on prefabrication (Gerbo et al., 2016b). An extensive, four-level geometric study was
424 performed to select ideal connection parameters, resulting in the following conclusions:

- 425 • Larger member angles reduce the allowable gap between members. In this study member
426 angles (α) up to 17.5° were found to result in feasible geometries.
- 427 • The versatility of a connection (*i.e.*, range of feasible parameters within a connection) can
428 be increased by using (1) members with thinner flanges [*e.g.*, t_m less than 25.4 mm (1.0
429 in.)], (2) longer plates [*e.g.*, $l_1 = l_2$ greater than 432 mm (17 in.)], or (3) longer slots [*e.g.*,
430 $d_{mh} = 47.6$ mm (1.875 in.)].
- 431 • Increasing the plate angle will probably decrease the versatility of a connection. This re-

432 relationship has minimal effect at low plate angles (*i.e.*, $\gamma = \beta = 0^\circ$ and 5°), with a more
433 significant effect at higher angles (*i.e.*, $\gamma = \beta = 10^\circ$ and 15°).

- 434 • The considered radii [$r_t = r_b = 63.5$ mm and 102 mm (2.50 in. and 4.00 in. respectively)]
435 of curvature do not play a significant role in the versatility of a connection.

436 Using selected connection parameters based on these geometric analyses, a total of 13 different
437 full-scale connection scenarios were tested to understand the impact of the (1) bolt tightening
438 procedure, (2) amount and direction of field bending, and (3) plate angle on the surface strains
439 induced during field installation. Strains were measured using DIC to capture full-field behavior.
440 Based on these experimental tests, the following conclusions and recommendations are made:

- 441 • The preferred bolt tightening procedure features one full turn per tightening increment
442 in a repeated criss-cross pattern to tighten the four-bolt connection. Tightening in larger
443 increments resulted in noticeable gouging or fracture of the bolts. Tightening in circular
444 patterns resulted in more asymmetry of strain patterns.
- 445 • The authors recommend limiting connections to $\delta = \pm 2.5^\circ$ as they were found to minimize
446 induced strain, representing a reasonable limit to fabrication tolerances for bent plates.
447 Residual strains from prefabrication must also be considered.
- 448 • For plates where the field bend increased the angle of the plates, the peak strains typically
449 occurred in the top plate near the point of contact with the member. For plates where the
450 field bend decreased the angle of the plates, the peak strains typically occurred within the
451 pre-bent region of the bottom plate.
- 452 • High strains were observed in the net section area of the bottom plate during the tightening
453 process for plates where the field bend increased the angle of the plates. For plates where
454 the field bend decreased the angle of the plates, high strains occurred in the net section
455 area of the top plate. These regions require additional attention during design as there
456 is enhanced potential for reduced ductility and fatigue resistance of the steel in the cold-
457 worked region. For connections subject to high cycle fatigue, this may limit acceptable

458 field bending angles (δ).

- 459 • The peak induced strain during field bending depends primarily on δ . Only varying the
460 plate angle had negligible effect induced on strains.

461 These measured results using DIC have provided an unprecedented understanding of this field
462 installation procedure. Additionally the results are relevant to cold bent double shear connections
463 in general and also useful for assessing their behavior and setting bend tolerances for fabrication.
464 Future work will focus on understanding the behavior of these connections under design and ser-
465 vice loads, including fatigue performance and both numerical and experimental research.

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543 **List of Tables**

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546 2 Summary of connection scenario parameters. * Abbreviations for bolt tightening
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548 (cw) = clockwise (1-4-2-3), (ccw) = counter-clockwise (4-1-3-2). 27

Symbol	Definition	Values Investigated
Plate Geometry		
γ	Top plate angle	0°, 5°, 10°, and 15°
β	Bottom plate angle	$\beta=\gamma$
l_1	Top plate length	305 mm - 559 mm (25.4 mm increments) [12 in. - 22 in. (1 in. increments)]
l_2	Bottom plate length	$l_2=l_1$
r_t	Top plate radius	63.5 mm, 102 mm [2.5 in., 4 in.]
r_b	Bottom plate radius	$r_b=r_t$
t_s	Plate thickness	12.7 mm (0.5 in.)
d_{ph}	Plate hole length	Oversized holes: 23.8 mm (0.9375 in.)
l_3	Plate hole end distance	76.2 mm (3 in.)
Member Geometry		
t_m	Member thickness	12.7 mm - 38.1 mm (3.18 mm increments) [0.5 in. - 1.5 in. (0.125 in. increments)]
d_m	Member depth	203 mm, 254 mm, and 305 mm (8 in., 10 in., and 12 in.)
d_{mh}	Member hole length	Oversized holes: 23.8 mm (0.9375 in.) Short slots: 25.4 mm (1 in.) Long slots: 47.6 mm (1.875 in.)
l_4	Member hole end distance	76.2 mm (3 in.)
Connection Configuration		
α	Member angle	$\alpha = \gamma \pm 5^\circ$ (0.5° increments)
g	Gap between members	$(g \geq d_m \sin \alpha + e) - \max(0.794 \text{ mm increments, } e=3.18 \text{ mm})$ (0.03125 in. increments, $e=0.125$ in.)
d_b	Bolt diameter	19.1 mm (0.75 in.)

TABLE 1. Geometric parameters for connection, including values for parametric investigation. See Figure 2A and 2B.

	$\gamma = \beta$ (deg.)	α (deg.)	δ (deg.)	$l_1=l_2$ (mm)	Tightening Procedure increment (pattern*)
1	10	17.5	7.5	483	1 turn/bolt (x)
2	10	17.5	7.5	483	3 turns/bolt (x)
3	10	17.5	7.5	483	Fully tighten bolt (x)
4	10	17.5	7.5	483	1 turn/bolt (cw)
5	10	17.5	7.5	483	1 turn/bolt (ccw)
6	10	12.5	2.5	483	1 turn/bolt (x)
7	10	7.5	-2.5	483	1 turn/bolt (x)
8	10	2.5	-7.5	483	1 turn/bolt (x)
9	0	2.5	2.5	381	1 turn/bolt (x)
10	5	2.5	-2.5	432	1 turn/bolt (x)
11	5	7.5	2.5	432	1 turn/bolt (x)
12	15	12.5	-2.5	533	1 turn/bolt (x)
13	15	17.5	2.5	533	1 turn/bolt (x)

TABLE 2. Summary of connection scenario parameters. * Abbreviations for bolt tightening procedure, with indications for bolt number (Figure 7): (x) = criss-cross (1-2-3-4), (cw) = clockwise (1-4-2-3), (ccw) = counter-clockwise (4-1-3-2).

549 **List of Figures**

550 1 Field bending: (A) rendering of initial un-tightened connection; (B) rendering of
551 final tightened connection; (C) photograph of initial un-tightened connection (Sce-
552 nario 1); (D) photograph of final tightened connection (Scenario 1). 30

553 2 Geometric definition of the connection, including definition of parameters and re-
554 lated variables for (A) flat plate and (B,C) bent plates (configuration exaggerated
555 to show dimensions), (D) definition of locations A-L, including example vectors
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558 3 Geometric analysis results, including (A) representative Level 1 analysis consider-
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565 4 Rendering of idealized geometry (left) and photograph of the experimental setup
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567 5 Experimental test setup shown for Scenario 1, including (A) elevation view, (B)
568 bolt tightening tools, and (C) instrumentation support system. 34

569 6 Measured engineering stress-strain relationships for a representative plate and bolt.
570 Plate data reprinted from Journal of Constructional Steel Research, 127, EJ Gerbo,
571 AP Thrall, BJ Smith, and TP Zoli, Full-field Measurement of Residual Strains in
572 Cold Bent Steel Plates, 187-203, 2016, with permission from Elsevier. 35

573 7 Longitudinal lines for data identification. Numbers indicate bolt identification. . . . 36

574 8 Effect of bolt tightening procedure: Measured circumferential surface strain (ϵ_x)
575 along lines A-D (Figure 7) for Scenarios 1, 2, 4, and 5 (Table 2). 37

576	9	Effect of bolt tightening procedure: Plan view of measured circumferential surface strain (ϵ_x). The left column indicates the evolution of strains during tightening (Scenario 1). The right column compares tightening patterns (Scenarios 1, 4, and 5, Table 2). All use 1 full turn of each bolt per tightening increment. Numbers indicate bolt identification.	38
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581	10	Effect of amount and direction of bend: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 1, 6, 7, and 8 (Table 2). The left column indicates strain induced during field bending, the right column shows total cumulative strain from prefabrication and field bending.	39
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585	11	Effect of amount and direction of bend: Plan view of measured circumferential surface strain (ϵ_x) for Scenarios 1, 6, 7, and 8 (Table 2).	40
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587	12	Effect of varying plate angles: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 9, 10, and 11 (Table 2).	41
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589	13	Effect of varying plate angles: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 12 and 13 (Table 2).	42
590			

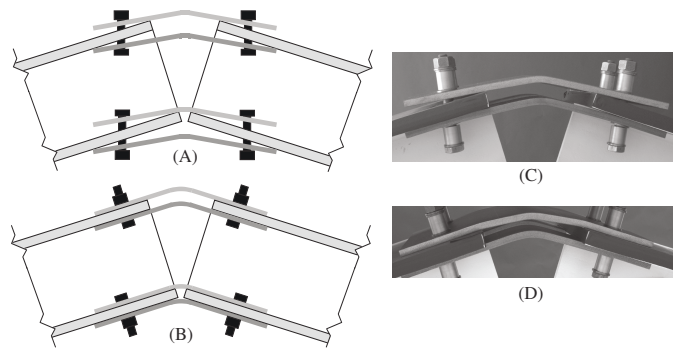


FIG. 1. Field bending: (A) rendering of initial un-tightened connection; (B) rendering of final tightened connection; (C) photograph of initial un-tightened connection (Scenario 1); (D) photograph of final tightened connection (Scenario 1).

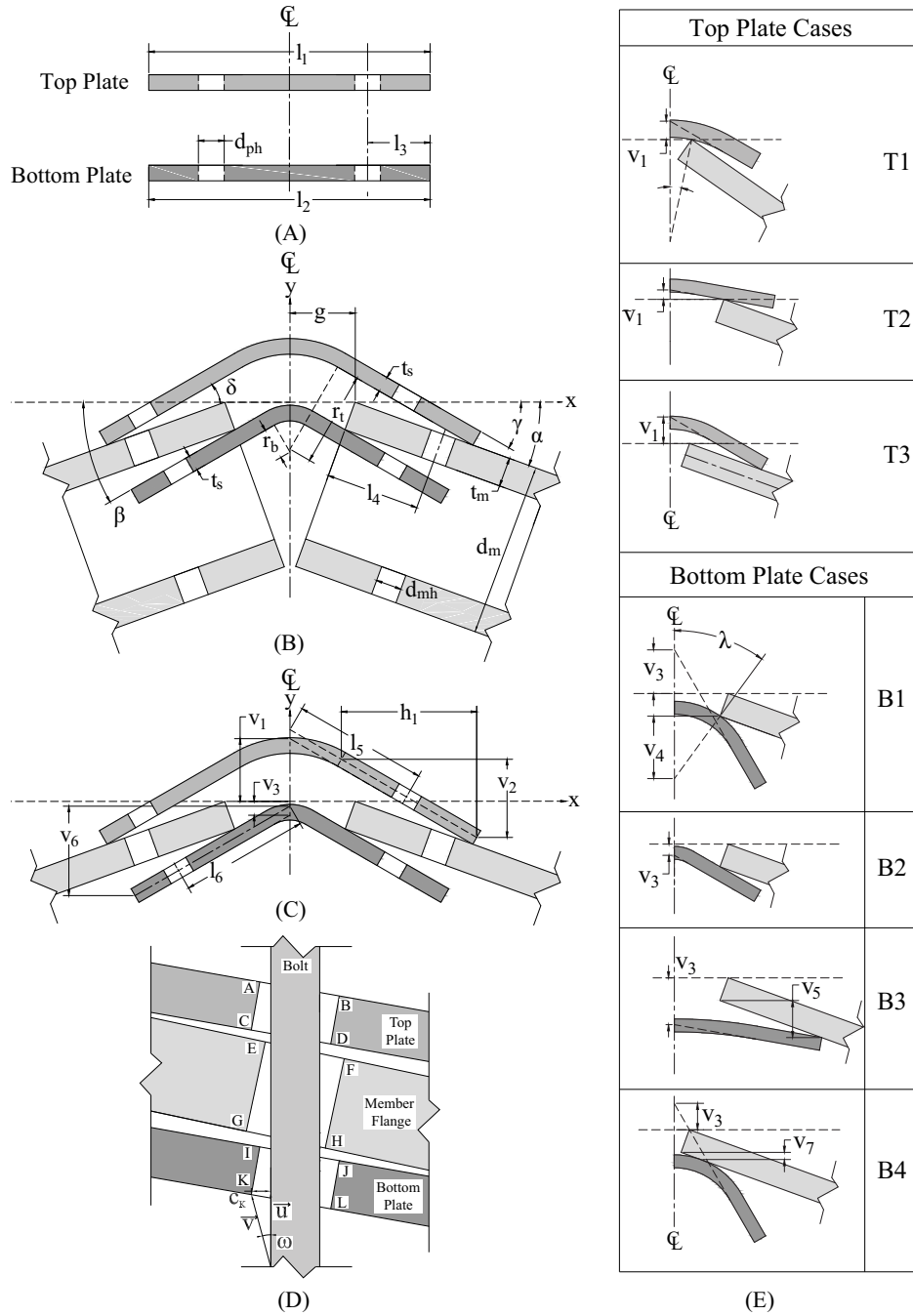
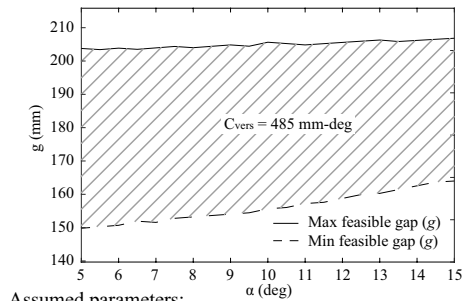
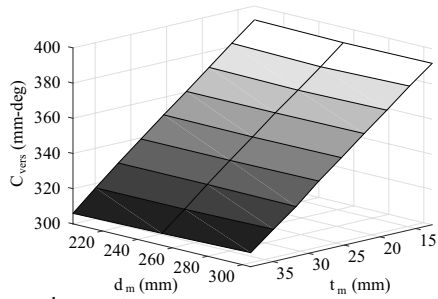


FIG. 2. Geometric definition of the connection, including definition of parameters and related variables for (A) flat plate and (B,C) bent plates (configuration exaggerated to show dimensions), (D) definition of locations A-L, including example vectors \vec{u} , \vec{v} and angle ω are shown for c_K , and (E) contact case definitions [only one plate (dark gray) and the member (light gray) are shown for simplicity].



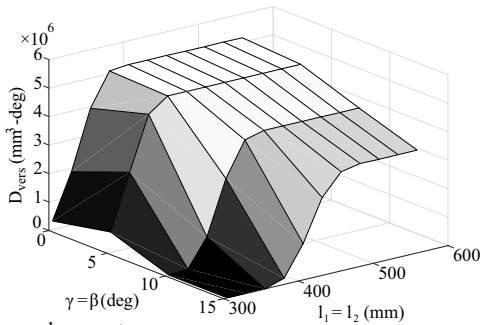
Assumed parameters:
 $t_m=25.4$ mm (1.0 in.), $d_m=254$ mm (10 in.),
 $\beta=\gamma=10^\circ$, $l_1=l_2=483$ mm (19 in.), $d_{mh}=47.6$ mm (1.875 in.),
 $r_b=r_t=102$ mm (4.0 in.), $d_{ph}=23.8$ mm (0.9375 in.)

(A)



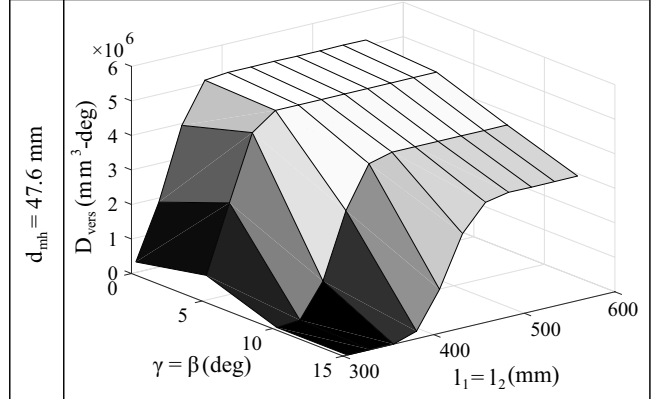
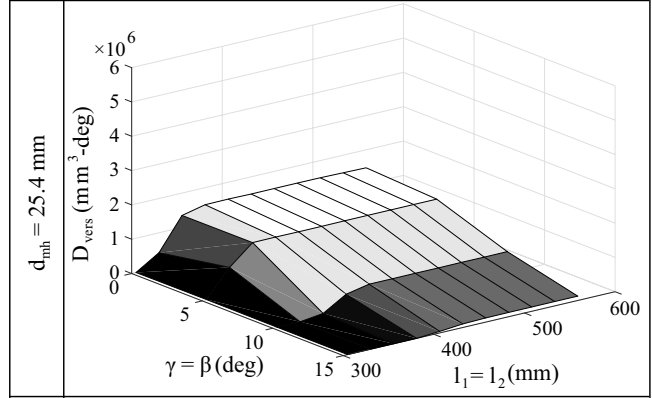
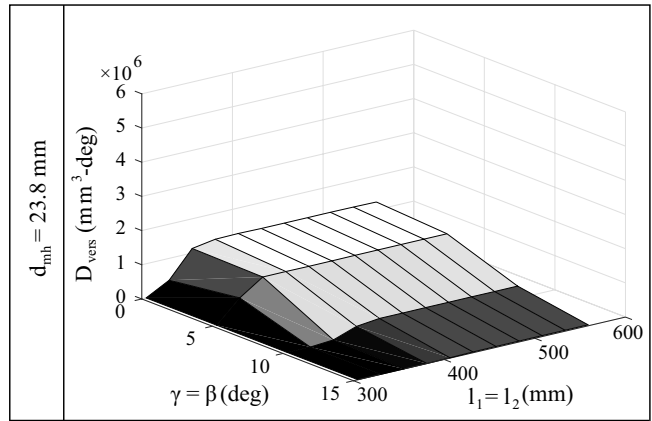
Assumed parameters:
 $\beta=\gamma=10^\circ$, $l_1=l_2=483$ mm (19 in.), $d_m=47.6$ mm (1.875 in.),
 $r_b=r_t=102$ mm (4.0 in.), $d_{ph}=23.8$ mm (0.9375 in.)

(B)



Assumed parameters:
 $d_{mh}=47.6$ mm (1.875 in.), $r_b=r_t=102$ mm (4.0 in.),
 $d_{ph}=23.8$ mm (0.9375 in.)

(C)



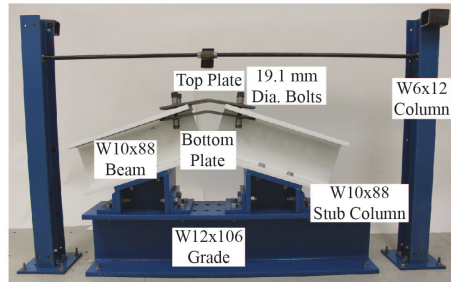
Assumed parameters:
 $r_b=r_t=102$ mm (4.0 in.), $d_{ph}=23.8$ mm (0.9375 in.)

(D)

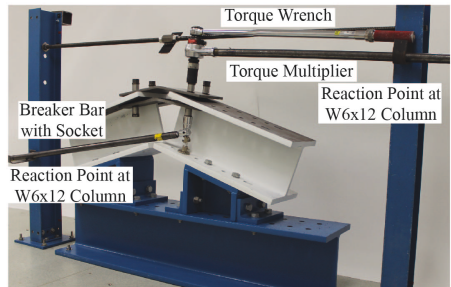
FIG. 3. Geometric analysis results, including (A) representative Level 1 analysis considering connection angle (α) and gap (g), (B) representative Level 2 analysis considering member depth (d_m) and member thickness (t_m), (C) representative Level 3 analysis considering plate length ($l_1 = l_2$) and initial plate angle ($\beta = \gamma$), and (D) representative Level 4 analysis considering member hole type (d_{mh}). Note: the color scale is proportional to the versatility metric on the vertical axis for (B), (C), and (D).

	Idealized	Experimental
Scenarios 1-5		
Scenario 6		
Scenario 7		
Scenario 8		
Scenario 9		
Scenario 10		
Scenario 11		
Scenario 12		
Scenario 13		

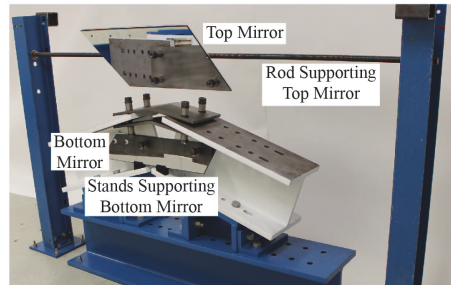
FIG. 4. Rendering of idealized geometry (left) and photograph of the experimental setup (right) for each tested Scenario.



(A)



(B)



(C)

FIG. 5. Experimental test setup shown for Scenario 1, including (A) elevation view, (B) bolt tightening tools, and (C) instrumentation support system.

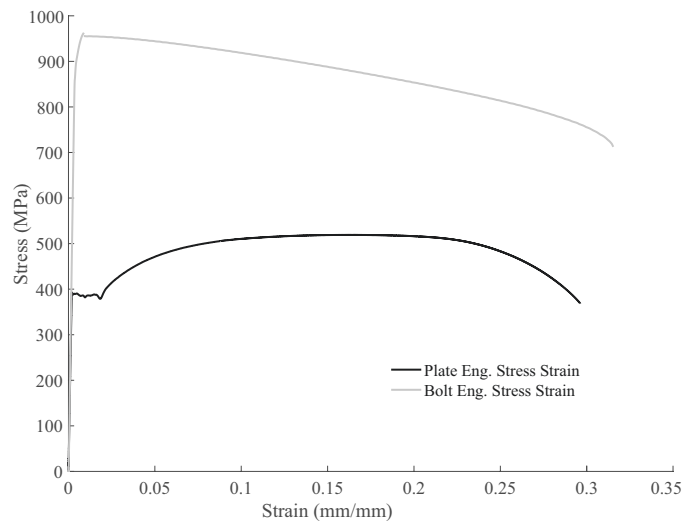


FIG. 6. Measured engineering stress-strain relationships for a representative plate and bolt. Plate data reprinted from *Journal of Constructional Steel Research*, 127, EJ Gerbo, AP Thrall, BJ Smith, and TP Zoli, Full-field Measurement of Residual Strains in Cold Bent Steel Plates, 187-203, 2016, with permission from Elsevier.

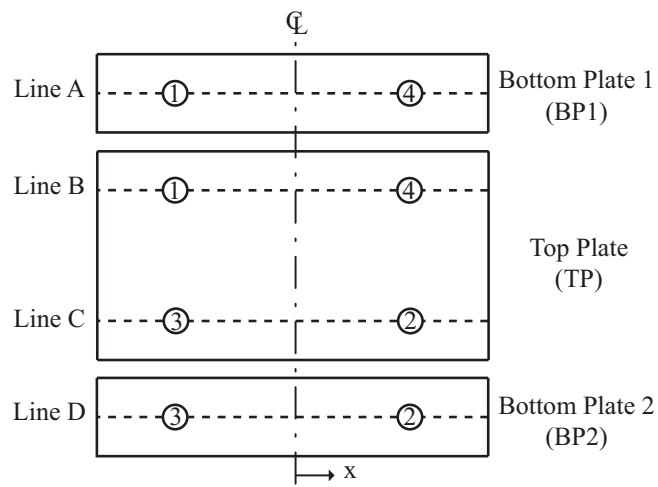


FIG. 7. Longitudinal lines for data identification. Numbers indicate bolt identification.

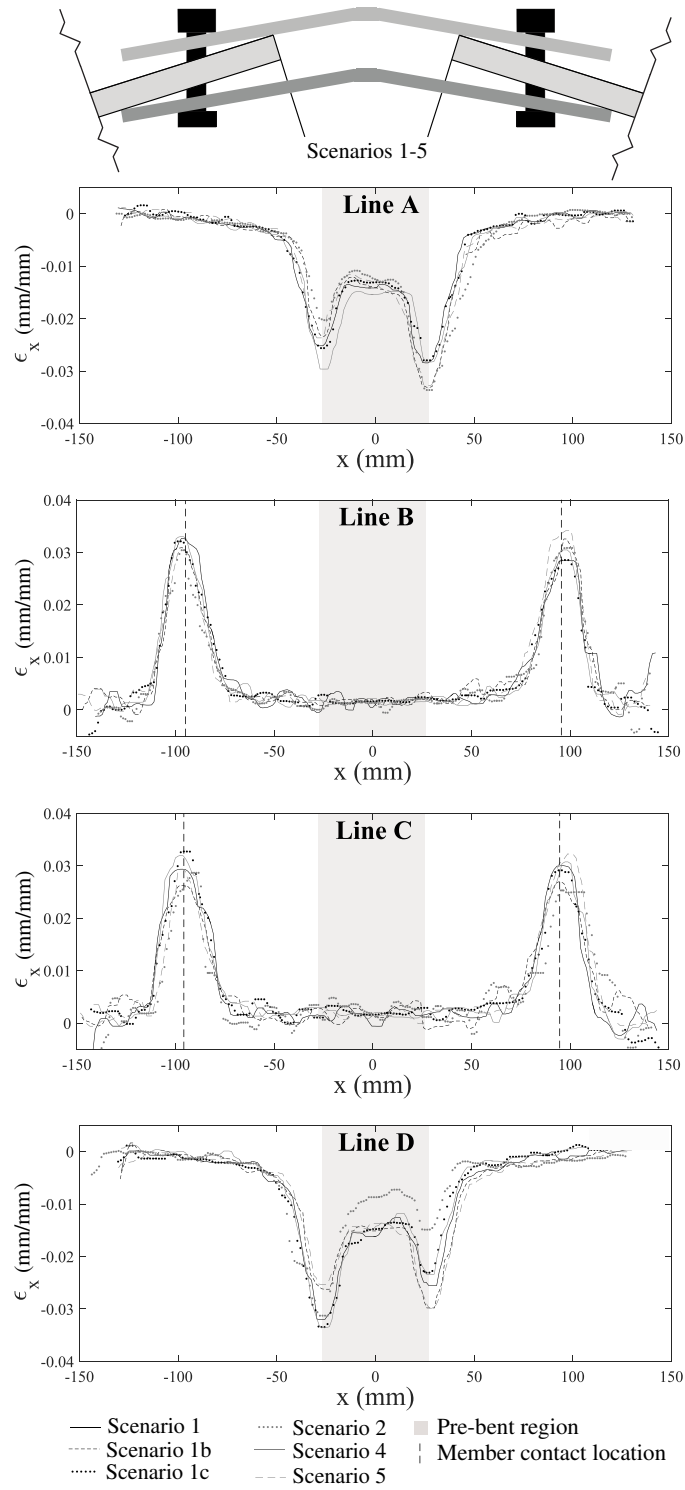


FIG. 8. Effect of bolt tightening procedure: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 1, 2, 4, and 5 (Table 2).

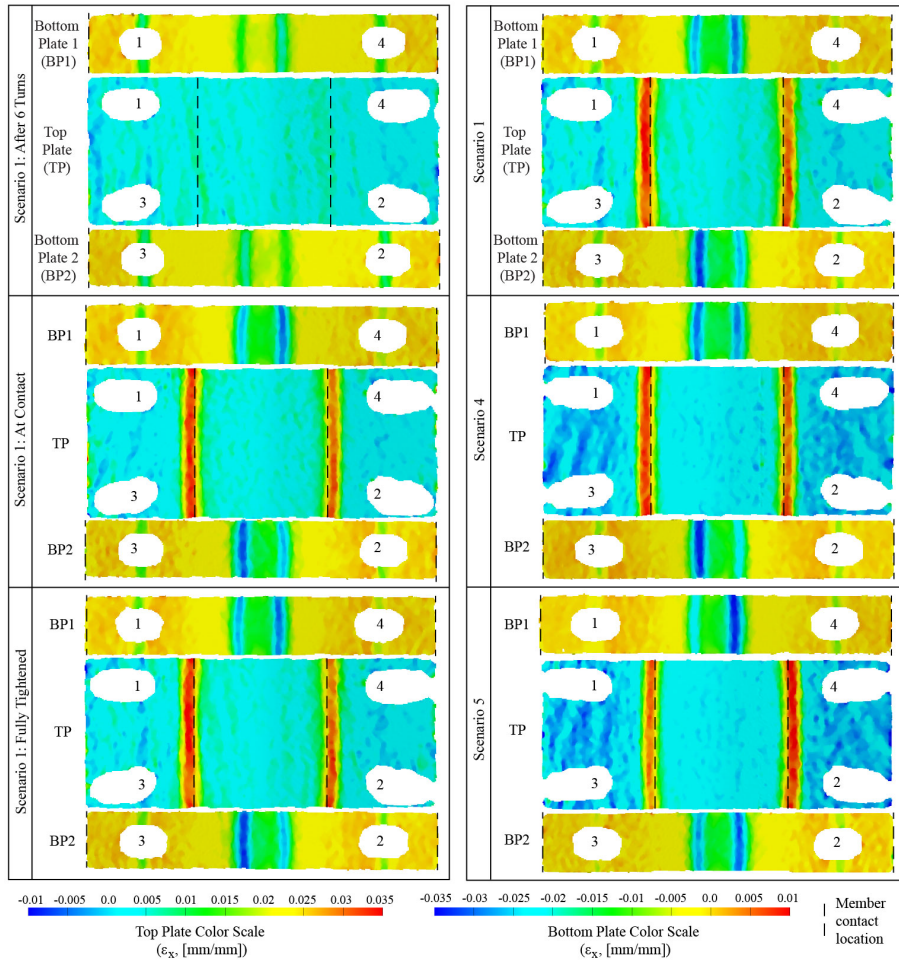


FIG. 9. Effect of bolt tightening procedure: Plan view of measured circumferential surface strain (ϵ_x). The left column indicates the evolution of strains during tightening (Scenario 1). The right column compares tightening patterns (Scenarios 1, 4, and 5, Table 2). All use 1 full turn of each bolt per tightening increment. Numbers indicate bolt identification.

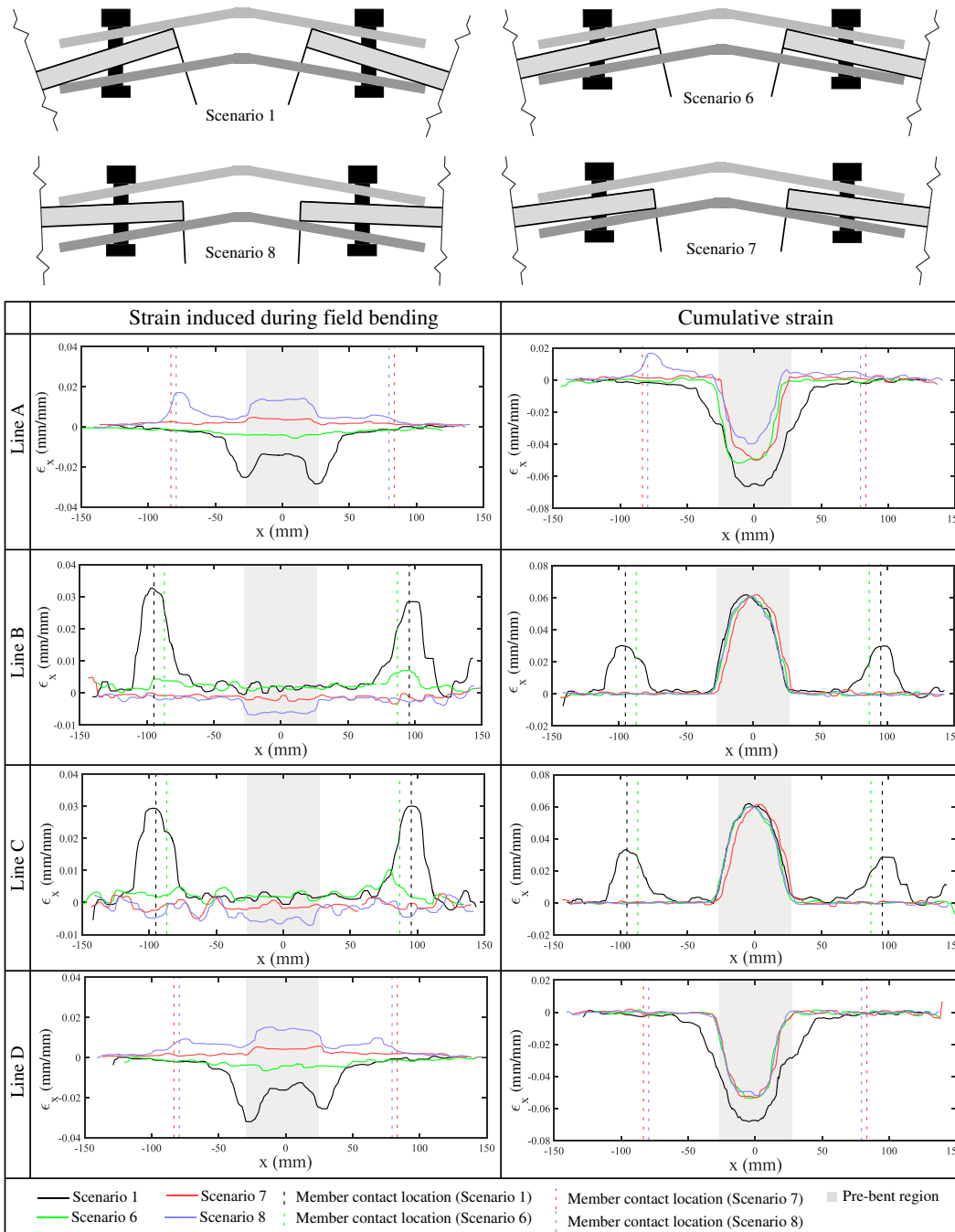


FIG. 10. Effect of amount and direction of bend: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 1, 6, 7, and 8 (Table 2). The left column indicates strain induced during field bending, the right column shows total cumulative strain from prefabrication and field bending.

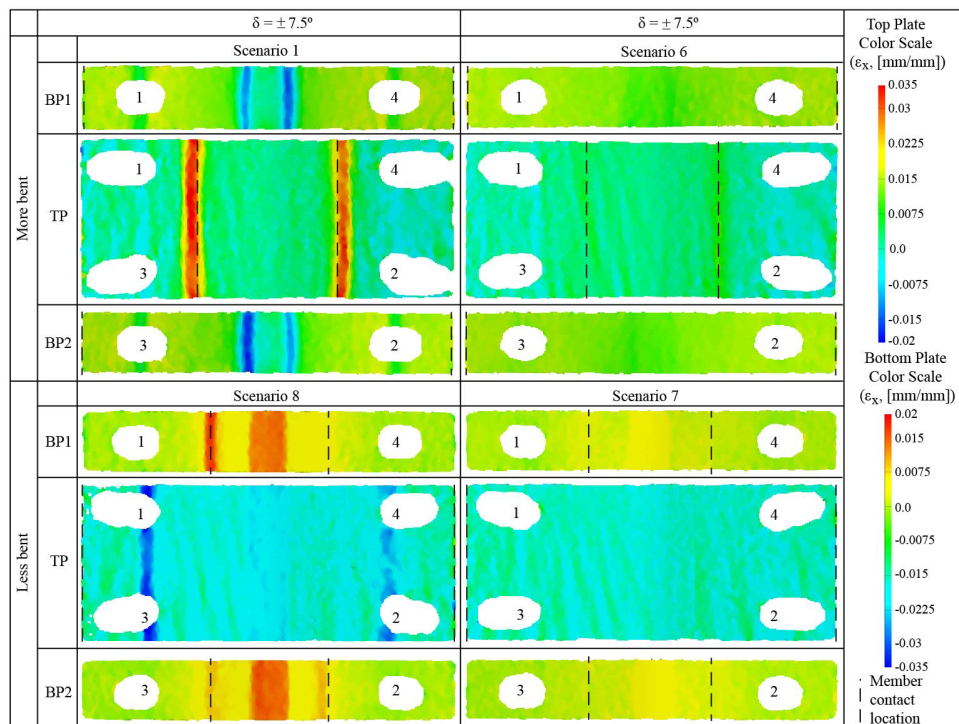


FIG. 11. Effect of amount and direction of bend: Plan view of measured circumferential surface strain (ϵ_x) for Scenarios 1, 6, 7, and 8 (Table 2).

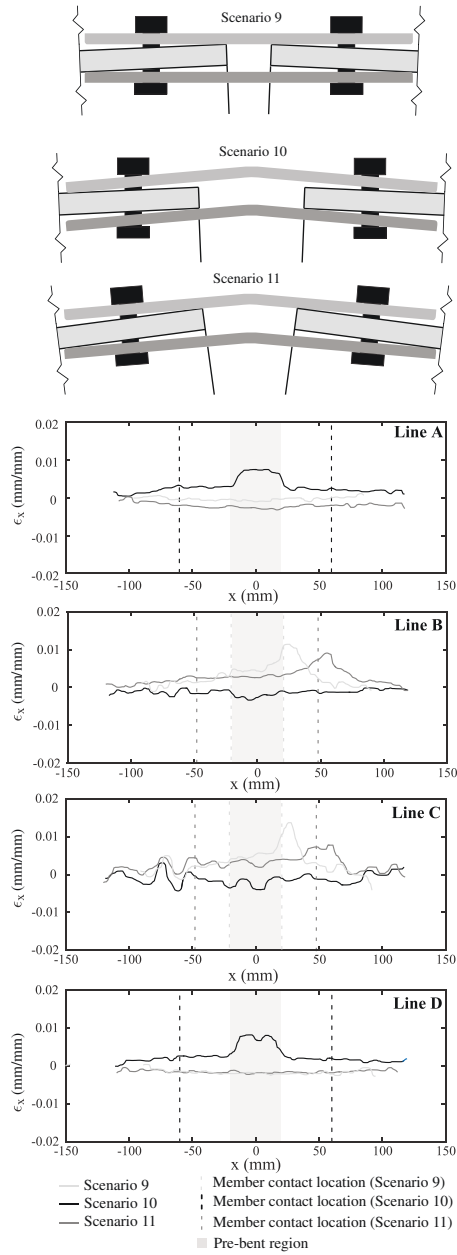


FIG. 12. Effect of varying plate angles: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 9, 10, and 11 (Table 2).

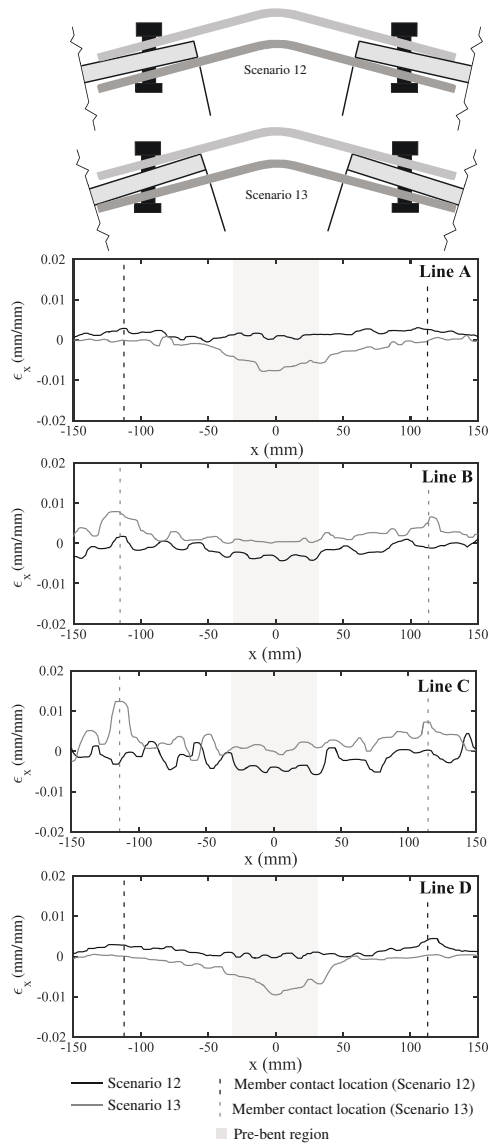


FIG. 13. Effect of varying plate angles: Measured circumferential surface strain (ϵ_x) along lines A-D (Figure 7) for Scenarios 12 and 13 (Table 2).